

STEAM TURBINES

If now A is the nozzle area in square feet, and W the quantity of steam passed in pounds per second,

$$r_{co} = \frac{WV_2}{A \sqrt{2g(P_1 - P_2)}}$$

since $PA_1^{1/n} = P_2 A_2^{1/n}$

Let $y = \frac{W}{A} = \frac{C \sqrt{P_1 - P_2}}{A}$ then this expression, giving the weight passed per unit of area, reaches a maximum for some definite value of A .
Writing

$\frac{dy}{dA} = 0$, when y is a maximum;

whence

$$\frac{1}{n} = \frac{2}{n+1} \quad \text{at the maximum point}$$

$\therefore \frac{P_2}{P_1} = \left(\frac{n}{n+1}\right)^{\frac{n+1}{n}}$ critical pressure ratio.

For superheated steam the value of n is 1.3, and for saturated steam 1.135. These values give a critical pressure ratio of 0.546 for superheated steam, and 0.577 for saturated steam.

Inserting the value of the critical drop in the formula for discharge, a simple expression for discharge per square inch of nozzle area is obtained:

$$(3) \quad W = \frac{C A \sqrt{P_1}}{a}$$

where a is the nozzle area in square inches;
 P_1 , the initial pressure in pounds per square inch;
 V_1 , the initial specific volume of steam in cubic feet per pound;
 k , a constant; 0.315 for superheated steam, and 0.3 for saturated steam.

Many experiments have been carried out to ascertain the actual value of k as compared with the theoretical values stated above. In general, experimental evidence points to the fact that a value of $k = 0.31$ may be attained in a well proportioned nozzle, both for superheated and saturated steam.

This means that the discharge of saturated steam exceeds the theoretical